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" Design and Assessment of Vertical Axis Wind Turbine Farms "

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Outlines

- Introduction
- Motivation and Problem Definition
- Contribution to Knowledge
- Strategy for Problem Solution
- Results
- Conclusion
- References



Introduction

Nonrenewable and Renewable Energy Resources:





- Based on REN21's 2014 report, renewable energy contributed by 19% to global energy consumption and 22 % to electricity generation
- The annual available wind energy is 25-70 TW

Global Wind Energy Capacity

 "Of all the forces of nature, I should think the wind contains the largest amount of motive power "

"Abraham Lincoln (1860)"

 The Wind energy has the fastest growing renewable power capacity
"Renewables Global Report (2006-2012)"





 The total worldwide wind capacity installed was 360 GW at the end of 2014, this provided 4% of the global electricity demand
"World Wind Energy Association 2014 half year report (August 2014)"

Types of Wind Turbines:

Horizontal Axis Wind Turbines





- Tower shadow effects
- Counter weights
- Less stability



Two Blades:

- less stability
- Low strength to wind shocks

Three Blades:

- Higher strength to wind storms.
- Less effect of tower shadow.
- Produces high output

Vertical Axis Wind Turbines



Darrieus

H-Rotor

Savonius







Horizontal Axis Wind Turbines (HAWTs) Farms:

- An isolated HAWT has the highest power coefficient Cp (0.4–0.45) Compared to all wind turbines
- 74% of the wind manufacturers invest in HAWTs while only 18% adopt the Vertical Axis Wind Turbines (VAWTs).
- Almost, all the existing wind turbine farms consists of HAWTs
- In close proximity to neighboring turbines, HAWTs suffer from a reduced power coefficient cause by the effect of the wake of upstream turbines





Wake interactions limit Conventional Horizontal Axis Wind Turbine Farm performance



Wind Farm Power Density:

 The power density (P.D.) of a wind farm is defined as the total power it can generate per unit land area it occupies

P . D =	Power Output of Farm	
	Area of Farm	

Vertical Axis Wind Turbine Farms

Vertical axis wind turbines (VAWTs) are proposed as an **alternative** to the more commonly used horizontal axis wind turbines (HAWTs) due to the potential increase in **power density** that is possible with VAWTs.

Experimental researches at CalTech found that the power density of H-rotor VAWT farms can be increased up to 30 W/m² by optimizing the placement of the turbines that enables them to extract energy from adjacent turbines wakes



John O. Dabiri, "Potential order-of-magnitude enhancement of wind farm power density via counter-rotating vertical-axis wind turbine arrays," Journal Of Renewable And Sustainable Energy 3: 043104 (2011)

A Numerical study on two dimensional Savonius turbine clusters, showed that a mutual enhancement between closely arranged turbines occurs as a function of :

- Relative direction of rotation.
- Gap Distance between rotors (S).
- Relative phase angle between the rotors:

 $\varphi = \theta \mathbf{1} - \theta \mathbf{2}$



Xiaojing Sun, Daihai Luo, Diangui Huang and Guoqing Wu,"Numerical study on coupling effects among multiple Savonius turbines," Journal of Renewable and Sustainable Energy 4: 053107 (2012)

Motivation and Problem Definition

Motivation:

Previous studies showed that VAWTs in close separation distances mutually enhance their power coefficients resulting in higher power output for individual turbines and higher power density if arranged in a wind farm

Problem (1):

All the **previous** studies **did not extended the results** for efficient layouts of VAWTs to develop **larger efficient farms** that generate the highest possible power output keeping a high power density

Problem (2):

Using non isolated turbines in a farm will make the prediction of a farm performance to become a complicated job as each turbine will have a different performance

Contribution to Knowledge

Solution (1):

This study introduces the idea of the efficient VAWT cluster as a building unit for an efficient VAWT farm that generates the highest possible power using the mutual enhancement between individual turbines and has a high power density compared to isolated counter-part

Solution (2):

Using the efficient cluster as a building unit for the **development of efficient** and patterned VAWT farms which have the same geometric topology of the cluster provides the capability of predicting the performance of a farm having a geometric progression of the efficient cluster.









Case Study (I)

Savonius Wind Turbine

Savonius Wind Turbine

- The Savonius wind turbine was invented 1929
- It is has a Simple Construction
- It is a drag-type device
- Consists of two or three buckets
- It has self starting capability
- Low noise levels due to operation at low tips speed ratio
- Savonius turbines have low efficiencies 15-20%

A. Shigetomi, Y. Murai, Y. Tasaka, Y. Takeda, "Interactive flow field around two Savonius turbines," Renewable Energy 36: 536-545, (2011)





10/15/2015

Computational Domain:



Grid Generation:

The grid structure consists of a non conformal mesh with **unstructured triangular elements** generated using ANSYS meshing.

An inflation of 10 levels of quad. cells is imposed to account for the boundary layer with a maximum thickness of 1 mm to achieve a y+ < 1 as required by the transition SST turbulence model [36, 37].



Numerical Model Validation:

Three models has been checked:

- K-ε model
- K-ω SST model
- Transition SST model

The transition SST turbulence model shows closer agreement to the experiments data in the numerical results for:

- Static torque coefficient at different azimuth angles (θ)
- Power coefficient at different tip speed ratios (λ)

The maximum power coefficient: $\mathbf{Cp}_{max} = 0.23 \text{ at } \lambda = 1$





Velocity Contours: Single Savonius Turbine



Vorticity Contours: Single Savonius Turbine

Flow Pattern around a Single Savonius Rotor

The flow patterns around the rotor are identified and found comparable to the experimental data:

- Flow (I): Coanda flow (Attached to the advancing blade convex side).
- Flow (II): Dragging flow (from the advancing blade convex side to the returning blade concave side).
- Flow (III): Overlap flow though the overlapping area.
- Flow (IV): Stagnation flow from upstream to the returning blade convex side.
- $\blacktriangleright \frac{Flow (V):}{blade tip.}$ Shedding vortex at the advancing
- Flow (VI): Shedding vortex from the returning blade tip.

Nakajima, M., IIO, S., Ikeda, T., Performance of double-step Savonius rotor for environmentally friendly hydraulic turbine. Journal of Fluid Science and Technology 3, (2008)





Grid	No. of	Cm	Cm
Level	Cells	Transition SST	DES
1	752,361	0.1461	0.1806
2	1,999,508	0.1561	0.1909
3	4,204,210	0.1557	0.1901







3D Validation for a Single Savonius Turbine



Transition SST model in the 3D Solution under predicted the single turbine performance compared to experimental data, this results are in consistence with the literature review on 3D solutions

To obtain better rsults, **detached eddy simulation (DES)** model is used for the flow simulation, the obtained results are **closer to the experimental data** for different TSR

The max. difference between DES and experimental data is about 15%



Numerical Solution of Two Savonius Wind Turbine Clusters

The development of an efficient cluster requires the study of different possibilities of multiple turbine cluster

Two **Co-rotating** and **Counter-rotating** turbine clusters in parallel and oblique configurations are numerically simulated



Parallel Configurations

Oblique Configurations

Xiaojing, S., Daihai, L., Diangui, H., Guoqing, W. Numerical study on coupling effects among multiple Savonius turbines. Journal of Renewable and Sustainable Energy 4: 053107, 2012

1- Two Parallel Co-Rotating Savonius Turbines:



Rotor (1) has higher efficiency enhancement than Rotor (2) due to the direction of rotation

Cp_{avg} max is 0.3 at 0.2D gap distance enhancement of 30% higher than isolated turbines

Reaches isolated turbine performance placed at approximately 5-6 D

2- Two Parallel Counter-Rotating Savonius Turbines

2.4

1.6

12

Gap Distance Percentage of Rotor Diameter



0.24

0.22

0.2

٥

AverageI





Comparison Between Co- and Counter-Rotating Two Parallel Turbines



Gap Distance (%of Rotor Diameter)

case (B) for counter-rotating rotors at a relative phase angle 30° is the most efficient for two parallel counter-rotating Savonius rotors, where the inward buckets are advancing buckets



Velocity Contours: Two Parallel Counter-Rotating Savonius Turbines



Vorticity Contours: Two Parallel Counter-Rotating Savonius Turbines

3- Two Oblique Co-Rotating Savonius Turbine Clusters



4- Two Oblique Counter-Rotating Savonius Turbine Clusters







Gap Distance (% of Rotor Diameter)

Numerical Simulation of Three Turbine Savonius Clusters







Velocity Contours: Three Co-Rotating Turbine Cluster



Vorticity Contours: Three Co-Rotating Turbine Cluster

Three Turbine Cluster Performance Confirmation

Co-Rotating Three turbine Cluster has an enhancement in average power coefficient of 26 % compared to isolated turbines

	Isolated	Rotor	Rotor	Rotor
	Rotor	(1)	(2)	(3)
Power Coefficient (Cp)	0.23	0.26	0.327	0.289
Enhancement % Compared to Isolated Rotor		13%	42%	25%
Ratio Compared to Rotor (1)		1	1.23	1.07

The results show that the ratio 1:1.2:1 between the power coefficients of the three rotors is consistent for λ >0.6

We focus here on the flow field at λ >0.6 in order to observe the flow-inducing action of the turbine revolutions rather than the flow-stagnating action at lower tip-speed ratios


Development of Patterned Savonius Wind Turbine Farms

- The efficient co-rotating three turbine cluster is used as a building unit for an efficient wind turbine farm
- Using the same triangular topology of the three turbine cluster the farm is developed as a geometric progression of the cluster
- Numerical simulations of farms that consist of nine and twenty-seven turbines are performed to confirm the pattern and the same enhancement ratio of the three turbine cluster and the geometric progression





Development of Patterned Vertical Axis Wind Turbine Farm

Nine Savonius Wind Turbine Farm

The **nine turbine farm** is **triangular** and has the **same topology** of the three turbine **cluster**

Each vertex of the triangle has a three turbine cluster

The domain, solver setting and turbulence model are similar to that of the three turbine cluster simulation

The same grid topology is used, this results in a grid having a number of cells equal to 817,253 cells for the nine turbine farm simulation

The simulation time is about 32 hours



Numerical Results for the Nine Savonius Wind Turbine Farm

The ratio between Cp of the clusters (A:B:C) is 1:1.2:1

The ratio between Cp of individual turbines is (1:1.2:1)

The average power coefficient achieved by the developed nine turbine farm at λ =1 is 26% higher than that of isolated nine turbine farm

Cluster A	Isolate	d Rotor	Rotor (1)		Rotor (2)	Rotor (3)	
Power Coefficient (Cp)	0.	23	0.24		0.3	0.26	
Enhancement % Compared to Isolated Rotor			4%		30%	13%	
Ratio Compared to Rotor (1)			1		1.25	1.08	
Cluster B	Isolate	d Rotor	Rotor (4)		Rotor (5)	Rotor (6)	
Power Coefficient (Cp)	0.23		0.31		0.36	0.32	
Enhancement % Compared to Isolated Rotor			34%		56%	39%	
Ratio Compared to Rotor (4)			1		1.16	1.03	
Ratio Compared to Rotor (4) Cluster C	Isolate	d Rotor	1 Rotor	(7)	1.16 Rotor (8)	1.03 Rotor (9)	
Ratio Compared to Rotor (4)Cluster CPower Coefficient (Cp)	Isolate	d Rotor 23	1 Rotor 0.26	(7)	1.16 Rotor (8) 0.32	1.03 Rotor (9) 0.27	
Ratio Compared to Rotor (4) Cluster C Power Coefficient (Cp) Enhancement % Compared to Isolated Rotor	Isolate 0.	d Rotor 23	1 Rotor 0.26 13%	(7) 5	1.16 Rotor (8) 0.32 39%	1.03 Rotor (9) 0.27 17%	
Ratio Compared to Rotor (4)Cluster CPower Coefficient (Cp)Enhancement % Compared to Isolated RotorRatio Compared to Rotor (7)	Isolated	d Rotor 23	1 Rotor 0.26 13%	(7) 6	1.16 Rotor (8) 0.32 39% 1.23	1.03 Rotor (9) 0.27 17% 1.03	
Ratio Compared to Rotor (4) Cluster C Power Coefficient (Cp) Enhancement % Compared to Isolated Rotor Ratio Compared to Rotor (7)	Isolated	d Rotor 23 Clust	1 Rotor 0.26 13% 1 er (A)	(7) 5	1.16 Rotor (8) 0.32 39% 1.23 uster (B)	1.03 Rotor (9) 0.27 17% 1.03 Cluster (C)	
Ratio Compared to Rotor (4) Cluster C Power Coefficient (Cp) Enhancement % Compared to Isolated Rotor Ratio Compared to Rotor (7) Power Coefficient (Cp)	Isolate 0.	d Rotor 23 Clust 0.	1 Rotor 0.26 13% 1 er (A) 26	(7) 5 Cl	1.16 Rotor (8) 0.32 39% 1.23 uster (B) 0.33	1.03 Rotor (9) 0.27 17% 1.03 Cluster (C) 0.28	



Tip Speed Ratio (λ)

1.5



The nine turbine farm is patterned, the performance of the farm can be predicted by simulation of a three turbine cluster

Using the ratio 1:1.2:1 and the results of the three turbine cluster, the nine turbine farm efficiency is predicted and compared to the calculated results

The error in the predicted average power coefficient (Cp=0.3) of the farm compared to the calculated value (Cp=0.292) is less than 2%



Velocity Contours: Co-Rotating Nine Savonius Turbine Farm



Vorticity Contours: Co-Rotating Nine Savonius Turbine Farm

Twenty Seven Savonius Wind Turbine Farm Simulation

The 27 turbine farm is **triangular** and has the **same topology** of the three turbine **cluster**

Each vertex of the triangle has a nine turbine farm

The domain, solver setting and turbulence model are similar to that of the three turbine cluster simulation

The same grid topology is used, this results in a grid having a number of cells equal to 2,751,759 cells for the 27 turbine farm simulation

The simulation time is about 120 hours



45

Numerical Results for the Twenty-Seven Savonius Wind Turbine Farm

The ratio between Cp of individual turbines is (1:1.2:1)

The ratio between average Cp of the clusters is 1:1.2:1

The ratio between average Cp of the nine turbine farms I, II and III is 1:1.2:1

The average power coefficient achieved by the developed nine turbine farms at λ =1 is 39% higher than that of isolated nine turbine farm

	Total Power (watt)	Ratio Compared to Cluster (A)				
Cluster (A)	170		1			
Cluster (B)	208		1.22			
Cluster (C)	182	1.07				
	Total Power (watt)	Ratio Co	mpared to Clu	ster (D)		
Cluster (D)	214		1			
Cluster (E)	233		1.09			
Cluster (F)	226		1.05			
	Total Power (watt)	Ratio Compared to Cluster (A)				
Cluster (G)	181		1			
Cluster (H)	233		1.27			
Cluster (I)	195	1.06				
		Juston (I)	Cluster (II)	Cluster		

	Cluster (I)	Cluster (II)	Cluster (II)
Total Power (watt)	567	680	604
Ratio Compared to Cluster (I)	1	1.2	1.06



The average power achieved by the developed twenty-seven turbine farm is 0.32 at $\lambda = 1$



The 27 turbine farm is **patterned**, this means that the performance of the farm can be **predicted** by simulation of only three turbines.

Using the ratio 1:1.2:1 and the results of the simulation of the three turbine cluster, the 27 turbine farm efficiency is predicted and compared to the calculated results.

The error in the predicted average power coefficient (Cp=0.326) of the farm compared to the calculated value (Cp=0.325) is less than 0.4%

Power and Power Density of the Developed VS Isolated Farms (One meter Diameter and One meter Height Turbines)



	Power	(W/m)	Enhancement	Power D	Enhancement	
			In	(W/n	In	
	Developed	Isolated	Power	Developed	Isolated	Power Density
3 Turbines	182	145	26%	17	13	1.3 times
9 Turbines	554	435	26%	10	1.9	7 times
27 Turbines	1841	1304	39%	4.5	1.5	4 times

Conclusion

- Multiple Vertical axis wind Turbines arranged in closely configuration show enhancement in their performance compared to their isolated counter parts.
- The close arrangement enables to construct farms of higher power densities compared to conventional aligned isolated farms
- Two types of turbines VAWTS are numerically studied: Savonius and Darrieus turbines
- The enhanced performance is numerically studied for two parallel and oblique co-rotating and counter rotating configurations
- The numerical results are used to develop an efficient triangular three turbine cluster

- The performance of the Savonius three Turbine cluster is higher than three isolated turbines by 26%, and its power density is 1.5 times the isolated turbines at λ =1
- The performance of the Darrieus three Turbine cluster is higher than three isolated turbines by 30%, and its power density is 4 times the isolated turbines at λ =2.8
- The clusters performance is confirmed at different tip speed ratios, and the power ratio between the turbines is 1:1.2:1 for the Savonius cluster and 1:1.1:1 for the Darrieus cluster
- The efficient three turbine clusters are used to build efficient patterned Vertical axis wind turbine farms having the same geometric topology of the cluster and the same power ratio enhancement
- The pattern is confirmed by solving 9 and 27 Savonius turbine farms and a 9 turbine Darrieus farm

- The developed farms are patterned and have the same power ratios of the three turbine clusters
- The power density of the Savonius nine turbine farm is about 7 times a nine isolated turbine farm, and the power density of the twenty-seven turbine farm is 4 times a twenty-seven isolated turbine farm
- The power density of the Darrieus nine turbine farm is more than 13 times a nine isolated turbine farm
- The advantage of the patterned farm appears is the power ratio achieved by the clusters and confirmed by the farms
- This power ratio is used to predict the performance of larger farms with the same topology to save processing time and man power

- The developed farms have smaller structure size, wind farm signature, material and manufacturing costs than conventional HAWTs
- The developed farms are simpler in logistics of installation, operation and maintenance
- The developed farms are safer for birds, produce lower noise, and have a lower impact on radar signals
- A preliminary analysis of the mean velocity field around the turbine blades at different tip speed ratios shows a velocity distribution close to the structure of a Rankine vortex
- A future work will be considered for numerical modeling of the Darrieus turbine as a combination of the free stream with a Rankine vortex.
- The solution of adjacent vortices can be used to determine the performance of closely oriented Darrieus turbines and represented a farm by a group of vortices

Summary of Savonius Turbine Clusters (Single Turbine Cp= 0.23 at λ =1)

	Co-Rotating			Counter-Rotating					Farms		
	Two	Two Turbines			Two Turbines						
	Parallel	Obli	Oblique Three		Parallel Oblique			Three	Nine	27	
		Case	Case	Turbines	Case	Case	Case	Case	Turbines	Turbines	Turbines
		Α	В		Α	В	С	D			
Ср	0.3	0.26	0.32	0.29	0.26	0.3	0.35	0.25	0.3	0.3	0.32
Gap Distance	0.2D	1.0D	0.2D	2.2D	0.8D	0.8D	0.4D	1.6D	2.2D		

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Publications

Published Journal Papers:

 M. Shaheen , M. El-Sayed, and Shaaban Abdallah, "Numerical Study of Two-Bucket Savonius Wind Turbine Cluster" Journal of Wind Engineering & Industrial Aerodynamics.

Reviewers Comments:

In fact, the reviewer acknowledges the authors for doing very well in the manuscript.

Without any doubt, the authors tackle a highly interesting problem in the field of VAWTs and the numerical approach sounds extremely inviting.

It is my impression that this is a well written paper and is of a quality/standard worth publishing

Bending Journal and Conference Papers

- M. Shaheen and Shaaban Abdallah, "Efficient Clusters and Patterned Farms for Darrieus Wind Turbines," Journal of Renewable Energy
- M. Shaheen and Shaaban Abdallah, "Development of Efficient Savonius Vertical Axis Wind Turbine Clustered Farms," Journal of Sustainable Energy
- M. Shaheen and Shaaban Abdallah, "A Numerical Comparison Study between Co-Rotating and Counter-Rotating Savonius Wind Turbine Clusters " 3rd International Conference and Exhibition on Mechanical & Aerospace Engineering 2015, US, Los Angeles

Poster Presentation:

- M. Shaheen , M. El-Sayed, and Shaaban Abdallah, "Numerical Study of Two-Bucket Savonius Wind Turbine Cluster," 2nd International Conference and Exhibition on Mechanical & Aerospace Engineering. September 08-10, 2014 Philadelphia, USA
- M. Shaheen , and Shaaban Abdallah, "A Novel Clustered and Patterned Vertical axis Wind Turbine Farm ," Graduate Student Expo & Poster Forum 2015, University of Cincinnati.

Under Construction

 " Physical Insights for vortex modeling of Darrieus Vertical Axis Wind Turbines"

Awards

- "Award for Exemplary Poster Presentation in the Category of Physical Science and Engineering," Graduate Student Expo & Poster Forum 2015, University of Cincinnati.
- "R. T. Davis Award, A Graduate Student with Demonstrated Aptitude and Scholar ship in the Field of Computational Mechanics,". University of Cincinnati, Department of Aerospace Engineering and Engineering Mechanics.



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 - For his guidance and all the useful discussions and brainstorming sessions
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- PhD students
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- My wife Walaa Mazhar, She has been a constant source of strength and inspiration



Force Analysis on Darrieus Turbine



Variation in Angle of Attack Vs Tip Speed Ratio



Transition in VAWTs



Effect of Turbine Radius on The Power Coefficient



At low tip speed ratios there is no effect for the turbine radius on the Power Coefficient

Induced Speed Vs Azimuth Angle



The induced wind speed v_a is given by

 $v_a = v_{\infty}(1-a)$

with the reduction factor

$$a = X \cdot \frac{r \cdot \omega_{rated}}{v_{\infty}} |\sin(\varphi)|$$



and the Wilson-Factor

$$X = \frac{B \cdot c}{2r}$$

The lift (F_L) and the drag (F_D) force are given by:

$$F_L = C_L \cdot \frac{\rho}{2} \cdot v_{res}^2 \cdot A_{bl}$$

$$F_D = C_D \cdot \frac{\rho}{2} \cdot v_{res}^2 \cdot A_{bl}$$



$$F_N = F_L \cdot \cos(\alpha) + F_D \cdot \sin(\alpha)$$

$$F_T = F_L \cdot \sin(\alpha) - F_D \cdot \cos(\alpha)$$









Why the Savonius turbine cannot be modeled with simple momentum theory

The performance of a single Savonius turbine utilizes the interactive flow between the two blades:

- 1) The flow attachment to the convex surface of the advancing blade produces a low-pressure region above it to pull the blade in torqueadding direction, i.e. the lift effect at a low tip-speed ratio
- 3) A large vortex is slowly shed behind the returning blade, which produces low-pressure downstream of the advancing blade.

Second Row





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Vorticity equation

$$\frac{D\vec{\omega}}{Dt} = \frac{\partial\vec{\omega}}{\partial t} + (\vec{u}\cdot\nabla)\vec{\omega}$$

The circulation around a closed contour is given by

$$\Gamma = \int_{C} \vec{U} \cdot \vec{dl} .$$

The lift per unit span is related to the circulation by the Kutta-Joukowski theorem of lift, $L = \rho U_{\infty} \Gamma$,

2.2 Prandtl's Lifting Line Theory

According to the Prandtl lifting line theory [Von Mises, 1959], a finite wing may be represented by a single equivalent line of vorticity known as the "bound vortex," since it is in a sense bound to the wing. The strength of this vortex, $\Gamma(y)$, is related to the lift distribution along the wing by the Kutta-Joukowski relationship:

$$L(y) = \rho U \Gamma(y) . \qquad (2.3)$$
Two-dimensional flow equation

$$\begin{split} \frac{\partial u}{\partial t} + u \frac{\partial u}{\partial x} + v \frac{\partial u}{\partial y} &= -\frac{1}{\rho} \frac{\partial p}{\partial x} + \nu \left(\frac{\partial^2 u}{\partial x^2} + \frac{\partial^2 u}{\partial y^2} \right), \\ \frac{\partial v}{\partial t} + u \frac{\partial v}{\partial x} + v \frac{\partial v}{\partial y} &= -\frac{1}{\rho} \frac{\partial p}{\partial y} + \nu \left(\frac{\partial^2 v}{\partial x^2} + \frac{\partial^2 v}{\partial y^2} \right), \\ \frac{\partial u}{\partial x} + \frac{\partial v}{\partial y} &= 0, \end{split}$$

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Axial Induction Factor

The fraction by which the axial component of velocity is reduced is known as the axial induction factor (a).

If the free stream velocity is U_{∞} and the axial velocity at the rotor plane is U_1 , then the axial induction factor is,



The power W extracted by the wind turbine is related to the axial induction factor (a):

$$\dot{W} = \frac{1}{2} \rho A U_{\infty}^{3} 4a(1-a)$$

Tangential induction factor

The tangential induction factor (a') which is due to rotation of the flow in the wake.

$$a'=\frac{\omega}{2\Omega},$$

where Ω is the angular speed of the rotor and ω is the angular velocity at which the wake rotates.

Wall Functions vs. Near-Wall Model



Numerous experiments have shown that the near-wall region can be largely subdivided into three layers. In the innermost layer, called the "viscous sub-layer", the flow is almost laminar, and the (molecular) viscosity plays a dominant role in momentum

"wall functions" are used to bridge the viscosity-affected region between the wall and the fully-turbulent region.

It is known that the log-law is valid for $y^+ > 30$ to 60.

Dimensionless wall distance (y plus)

 u_*

y

 ν

$$y^+ \equiv \frac{u_* \, y}{\nu}$$

friction velocity at the nearest wall

the local kinematic viscosity of the fluid

the distance to the nearest wall

$$\operatorname{Re}_{x} = \frac{\rho U_{\infty} L}{\mu}$$
$$C_{f} = \frac{0.026}{\operatorname{Re}_{x}^{1/7}}$$

$$\tau_{\rm wall} = \frac{C_f \rho {U_{\infty}}^2}{2}$$

$$U_{\rm fric} = \sqrt{\frac{\tau_{\rm wall}}{\rho}}$$

wall spacing

$$\Delta S = \frac{y^+ \mu}{u_{\rm fric} \rho}$$

http://www.pointwise.com/yplus/

Intermittency Contours

ANSYS ANSYS Model based on Intermittency

- Intermittency:
- Laminar flow:
- Turbulent flow $\gamma = 1$
- Transition



 $\gamma = \frac{t_{turb}}{t_{lam} + t_{turb}}$

 $\gamma = 0$



Average Intermittency on the blades 0.4

For point surfaces, the value is interpolated from all the mesh nodes of the cell containing the point



 $t = \frac{500 \ \mu}{\rho U^2}$

Average Intermittency on the blades 0.9

Contours of Turbulence Intensity



K-w (SST model) full turbulent

Oct 14, 2014 ANSYS Fluent 14.5 (2d, pbns, sstkw, transient)

K-w (Trans SST model)





Magnification of Transition Position









Experimental Data Correction

An object placed in a wind tunnel produces some "tunnel blockage"

This causes the local wind velocity in the test section to increase.

This increase has to be accounted for by determining a tunnel blockage factor (ϵ)

The total tunnel blockage correction be determined by applying the following equation:

$$\epsilon = \frac{1}{4} \frac{\text{Model frontal area}}{\text{Test section area}}$$

Pope, A. and Harper, J.J., Low Speed Wind Tunnel Testing, John Wiley, New York, 1966.

Tip speed ratio

$$X_{\infty} = R\Omega / V_{\infty}$$

Torque coefficient

$$C_Q = (Q + Q_f) / \frac{1}{2} \rho_\infty V_\infty^2 R A_s$$

Power coefficient

$$C_P = (Q + Q_f) \Omega / \frac{1}{2} \rho_{\infty} V_{\infty}^3 A_s$$

$$V_{\infty} = V_{\infty_{u}} (1+\epsilon)$$

$$q_{\infty} = q_{\infty_{u}} (1+\epsilon)^{2}$$

My Status

- Arrived at USA on 23 September 2012
- Did written qualifier exam on November 2013
- Finished courses on December 2013, with GPA: 3.833
- Did oral qualifying exam on January 2014
- Did proposal defense on October 2014